

Thermal response

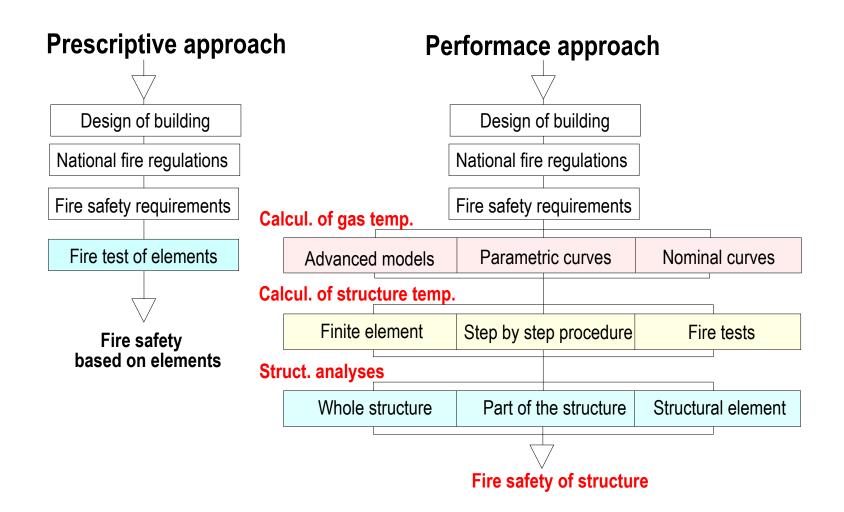
Raul ZAHARIA

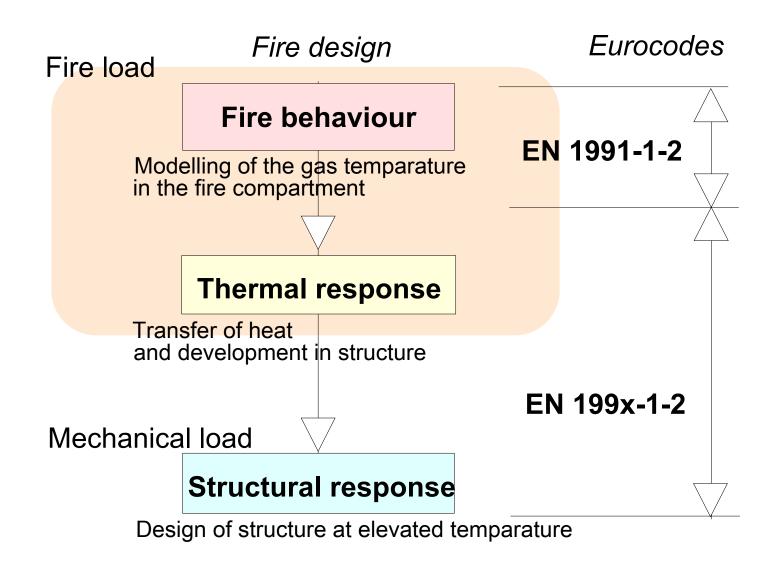
Lecture 3: 1/04/2014

European Erasmus Mundus Master Course

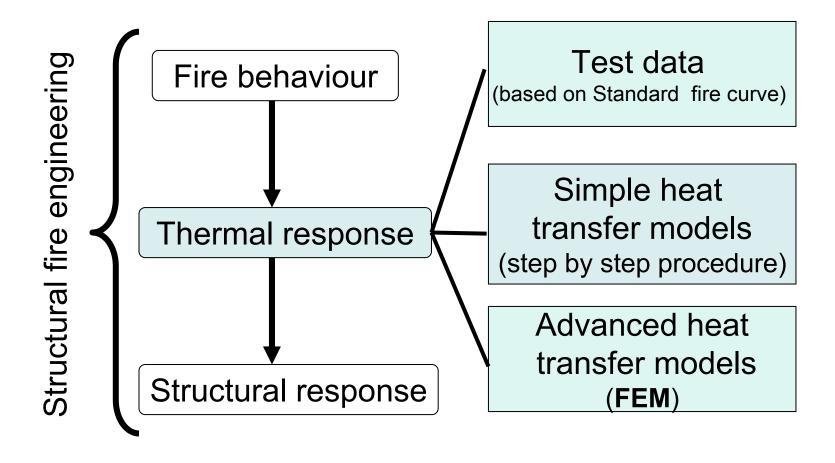
Sustainable Constructions under Natural Hazards and Catastrophic Events







Thermal response



Basics of heat transfer

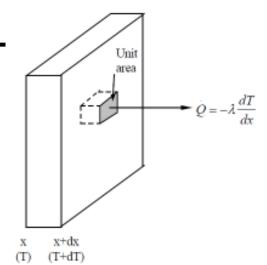
Three basic mechanisms

- CONDUCTION Heat transferred within an object or between two objects in contact
- CONVECTION Heat transferred by movement of currents within fluids or air
- RADIATION Transfer of energy by electromagnetic waves (does not require matter to transfer thermal energy)

Heat conduction

Fourier's law of heat conduction

$$\dot{\mathbf{Q}} = -\lambda \frac{dT}{dx}$$



dT - is the temperature difference across an infinitesimal thickness dx

 $\mathcal Q$ - is the rate of heat transfer (heat flux) across the material thickness (the amount of energy that flows through a unit area per unit time – W/m²)

 λ - is the conductivity (W/mK)

The minus sign in equation indicates that heat flows from the higher temperature side to the lower temperature side.

Radiant heat transfer of black body surfaces

The total amount of thermal radiation emitted by a black body surface is a function of its temperature only

$$E_{\rm b} = \sigma T^4$$

where

 σ is the Stefan-Boltzmann constant = 5.67x10⁻⁸ W/(m²K⁴)

T is the temperature in K

Radiant heat transfer of grey body surfaces

No real material emits and absorbs radiation according to laws of the black body.

The total radiant energy emitted by a general surface

$$E = \varepsilon \sigma T^4$$

where

- arepsilon is the emissivity
- σ is the Stefan-Boltzmann constant = 5.67x10⁻⁸ W/(m²K⁴)
- T is the temperature in K

Net heat flux

Thermal actions are given by the net heat flux to the surface of the element.

The net heat flux h_{net} is given as the sum of effects of convection and radiation

The design value is given by

$$h_{\text{net,d}} = h_{\text{net,c}} + h_{\text{net,r}}$$

where

 $h_{\text{net,c}}$ is the heat flux from convection [W/m²]

 $h_{\text{net,r}}$ is the heat flux from radiation [W/m²]

Convective heat flux

The net convective heat flux [W/m²] is given by

$$h_{\mathsf{net,c}} = lpha_{\mathsf{c}} ig(heta_{\mathsf{g}} - heta_{\mathsf{m}} ig)$$

where

 $\alpha_{\rm c}$ is the coefficient of heat transfer by conduction

- $\alpha_c = 25 \text{ W/m}^2\text{K}$ for standard curve
- $\alpha_c = 35 \text{ W/m}^2\text{K}$ for parametric curve
- $\alpha_c = 50 \text{ W/m}^2\text{K}$ for hydrocarbon curve
- $\alpha_c = 35 \text{ W/m}^2\text{K}$ for zone models and localized fires
- $\theta_{\rm g}$ is the gas temperature in the vicinity of the fire exposed element [°C]
- $\theta_{\rm m}$ is the surface temperature of the element [°C]

Radiative heat flux

The net radiative heat flux [W/m²] is given by

$$h_{\text{net,r}} = \phi \ \varepsilon_{\text{res}} \ 5,67 \cdot 10^{-8} \left[\left(\theta_r + 273 \right)^4 - \left(\theta_{\text{m}} + 273 \right)^4 \right]$$

where

 ϕ is the configuration factor, usually ϕ = 1,0 (a lower value may be considered, to take into account the shadow effects – see Annex G of EN1991-1-2)

 $\varepsilon_{\rm res}$ is the resulting emissivity, see next page

 $\theta_{\rm r}$ is the radiating temperature [°C], which can be taken equal to the gas temperature $\theta_{\rm g}$

 $\theta_{\rm m}$ is surface temperature of the element [°C]

5,67·10⁻⁸ is Stefan-Boltzmann constant [W/(m²K⁴)]

Resulting emissivity

The emissivity is changing during the fire, influenced by the amount of carbon particles and dust in the smoke, the colour and the temperature of the surface.

Its value has a significant effect on the accuracy of the solution

$$\varepsilon_{\rm res} = \varepsilon_{\rm f} \ \varepsilon_{\rm m}$$

where

 $\varepsilon_{\rm f}$ is the emissivity of the fire, usually $\varepsilon_{\rm f}$ = 1,0

 $\varepsilon_{\!_{m}}$ is the emisivity of the material surface

- carbon steel element, concrete $\varepsilon_{\rm m}$ = 0,7
- stainless steel element $\varepsilon_{\rm m}$ = 0,4
- aluminum alloys $\varepsilon_{\rm m} = 0.3$

Heat transfer

Equilibrium between the quantity of energy (heat) that penetrates in a body through its surface and the energy used to modify the temperature of the body in a given time period Δt

Increase of body temperature

$$mc\Delta\theta = A_{\rm m} h_{\rm net,d} \Delta t$$

The heat received on surface

or
$$\rho V c \Delta \theta = A_{\rm m} h_{\rm net,d} \Delta t$$
 $\Rightarrow \Delta \theta = \frac{A_{\rm m} / V}{\rho c} h_{\rm net,d} \Delta t$

where

m is the mass of the body

V volume of the body [m³]

A_m surface area of the body [m²]

c specific heat of body [J/kgK]

ho density of body [kg/m 3]

 $h_{\rm net,d}$ net heat flux received by the surface of the body [W/m²]

 Δt time period

Step by step procedure

for the fire unprotected steel elements in EN1993-1-2

4.2.5 Steel temperature development

4.2.5.1 Unprotected internal steelwork

(1) For an equivalent uniform temperature distribution in the cross-section, the increase of temperature $\Delta \theta_{a,t}$ in an unprotected steel member during a time interval Δt should be determined from:

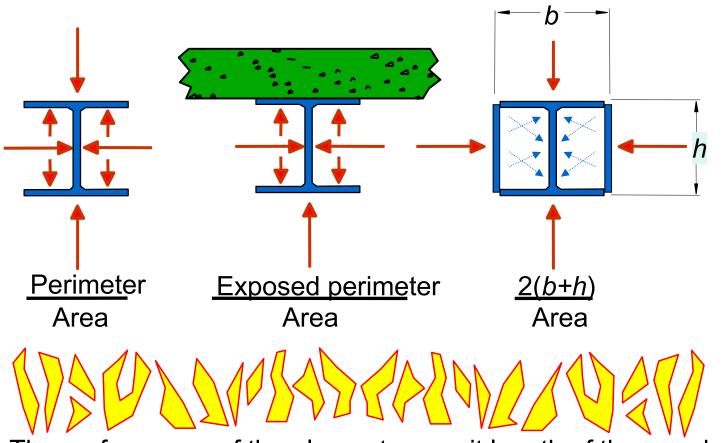
$$\Delta \theta_{a,t} = k_{sh} \frac{A_m/V}{c_a \rho_a} \dot{h}_{net,d} \Delta t \tag{4.25}$$

where:

```
k sh
                      correction factor for the shadow effect, from 5.2.5.1(2)
              18
A_{\rm m}/V
                      the section factor for unprotected steel members;
A_{
m m}
                      the surface area of the member per unit length [m<sup>2</sup>]:
               18
V
                      the volume of the member per unit length [m³];
               is
                      the specific heat of steel, from section 3 [J/kgK];
C_{\mathsf{a}}
               18
                      the design value of the net heat flux per unit area [W/m<sup>2</sup>];
               15
\Delta t
               is
                      the time interval [seconds];
                      the unit mass of steel, from section 3 [kg/m<sup>3</sup>].
               18
\rho_{\rm a}
```

Section factor $A_{\rm m}/V$

for the fire unprotected steel element



The surface area of the element per unit length of the member divided

by the volume of the member per unit length.

Table 4.2: Section factor A_m/V for unprotected steel members.

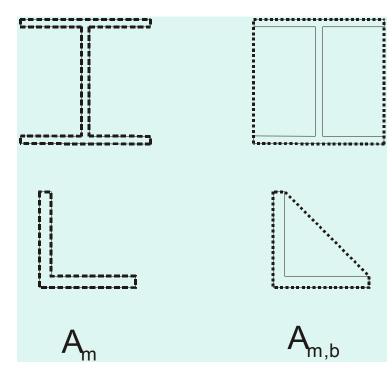
Open section exposed to fire on all sides: $\frac{A_m}{V} = \frac{\text{perimeter}}{\text{cross-section area}}$	Tube exposed to fire on all sides: $A_{\mathfrak{w}}/V = 1/t$ \downarrow \downarrow \downarrow
Open section exposed to fire on three sides: $\frac{A_{aa}}{V} = \frac{\text{surface exposed to fire}}{\text{cross-section area}}$	Hollow section (or welded box section of uniform thickness) exposed to fire on all sides: If $t \le A_m/V = 1/t$
I-section flange exposed to fire on three sides: $A_{\rm in}/V = (b+2t_{\rm f})/(bt_{\rm f})$ If $t < b$: $A_{\rm in}/V = 1/t_{\rm f}$	Welded box section exposed to fire on all sides: $\frac{A_{m}}{V} = \frac{2(b+h)}{\text{cross-section area}}$ If $t \le b$: $A_{m}/V = 1/t$
Angle exposed to fire on all sides: $A_{\rm m}/V=2/\tau$	I-section with box reinforcement, exposed to fire on all sides: $\frac{A_m}{V} = \frac{2(b+h)}{\text{cross-section area}}$
Flat bar exposed to fire on all sides: $A_{\rm m}/V = 2(b+t)/(bt)$ If $t \leqslant b$: $A_{\rm m}/V = 2/t$	Flat bar exposed to fire on three sides: $A_{\mathfrak{m}}/V = (b+2t)/(bt)$ If $t \leqslant b$: $A_{\mathfrak{m}}/V = 1/t$

Shadow effect

$$k_{sh} = \frac{\left[A_m/V\right]_b}{\left[A_m/V\right]} = \frac{1}{\left[A_m/V\right]}$$

The shadow effect takes into account that there cannot be more radiation reaching the surface of the member than the radiation travelling through the smallest box surrounding the section.

For temperatures normally encountered in a fire, the radiation is the dominant heat transfer mode to the elements.



Shadow effect

(2) For I-sections under nominal fire actions, the correction factor for the shadow effect may be determined from:

$$k_{sh} = 0.9 [A_m/V]_b/[A_m/V]$$
 (4.26a)

where:

 $[A_m/V]_b$ is box value of the section factor

In all other cases, the value of k_{sh} shall be taken as:

$$k_{sh} = [A_m/V]_b/[A_m/V]$$
 (4.26b)

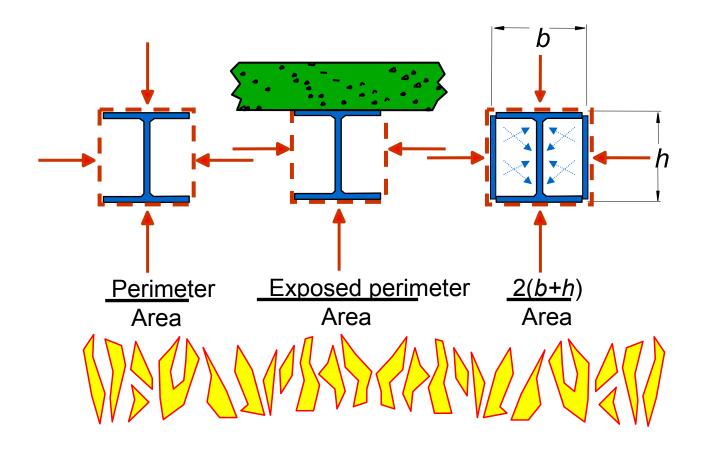
NOTE (1): For cross sections with a convex shape (e.g. rectangular or circular hollow sections) fully embedded in fire, the shadow effect does not play role and consequently the correction factor k_{sh} equals unity.

NOTE (2): Ignoring the shadow effect (i.e.: $k_{sh} = 1$), leads to conservative solutions.

- (4) The value of $\dot{h}_{net,d}$ should be obtained from EN 1991-1-2 using $\varepsilon_f = 1.0$ and ε_m according to 2.2(2), where ε_f , ε_m are as defined in EN 1991-1-2.
- (5) The value of Δt should not be taken as more than 5 seconds.
- (6) In expression (4.26) the value of the section factor $A_{\rm m}/V$ should not be taken as less than $10\,{\rm m}^{-1}$.

Section factor $(A_m/V)_b$

for shadow effect, unprotected steel element



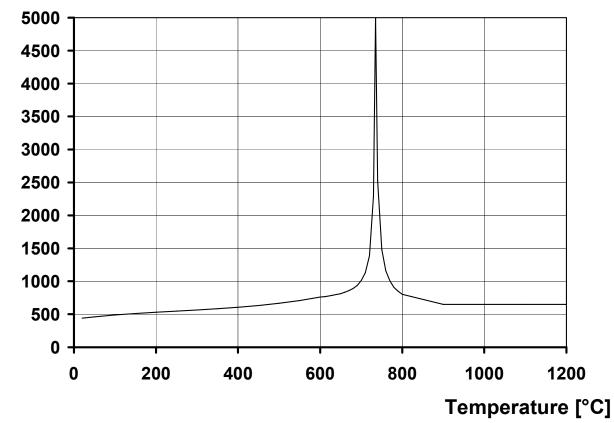
Thermal properties

of carbon steel at elevated temperatures

Specific heat

Model in EN1993-1-2:2005





for 20 °C
$$\leq \theta_{\rm a}$$
 < 600 °C $c_{\rm a}$ = 425 + 7,73 × 10⁻¹ $\theta_{\rm a}$ - 1,69 × 10⁻³ $\theta_{\rm a}$ ² + 2,22 · 10⁻⁶ $\theta_{\rm a}$ ³ J/kgK for 600 °C $\leq \theta_{\rm a}$ < 735 °C $c_{\rm a}$ = 666 + J/kgK for 735 °C $\leq \theta_{\rm a}$ < 900 °C $c_{\rm a}$ = 545 + J/kgK for 900 °C $c_{\rm a}$ = 650 J/kgK

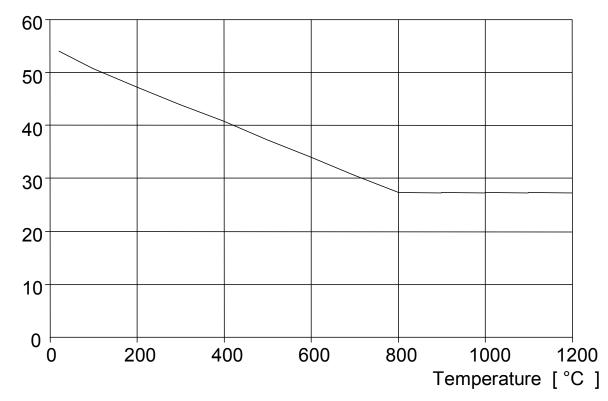
where:

 θ_a is the steel temperature [°C]

Thermal conductivity

Model in EN1993-1-2:2005

Thermal conductivity [W/mK]

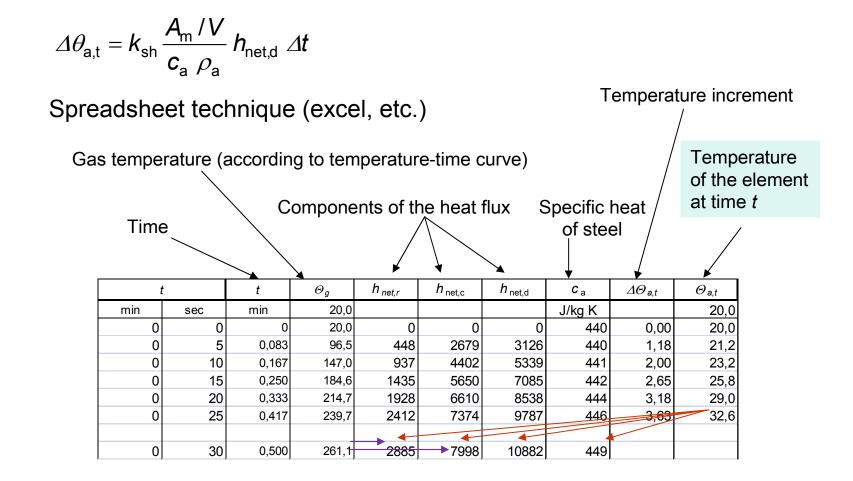


for 20 °C
$$\leq \theta_{\rm a}$$
 < 800 °C
 $\lambda_{\rm a}$ = 54 - 3,33 · 10⁻² $\theta_{\rm a}$ W/mK for 800 °C $\leq \theta_{\rm a} \leq$ 1200 °C
 $\lambda_{\rm a}$ = 27,3 W/mK

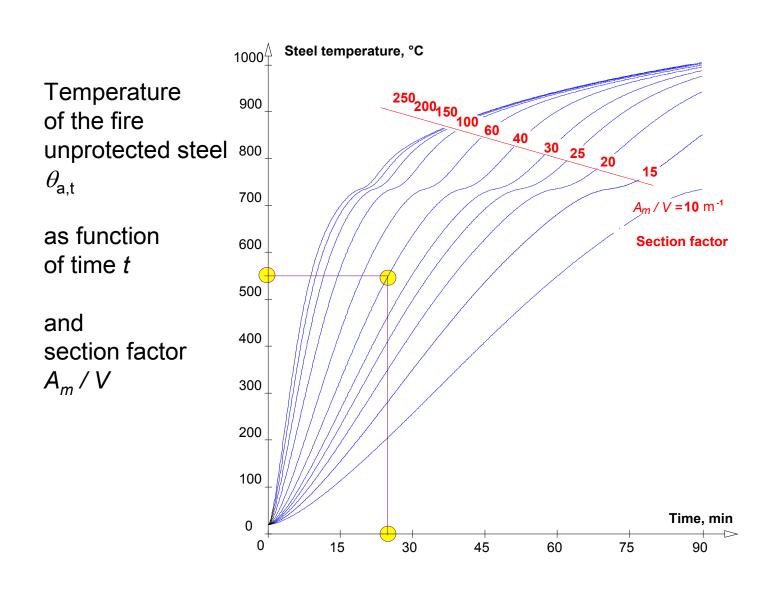
where:

 θ_a is the steel temperature [°C]

Technique of step by step precedure



Result of step by step precedure



Tables and graphs – unprotected steel

http://www.ct.upt.ro/users/RaulZaharia/Introduction fire design.pdf

A* _m /V [m ⁻¹]	400	200	100	60	40	25	
V/A* _m [mm]	2.5	5.0	10.0	16.7	25.0	40.0	
Time [min.]	Steel temperature in °C						
0	20	20	20	20	20	20	
5	430	291	177	121	90	65	
10	640	552	392	276	204	142	
11	661	587	432	308	228	159	
12	678	616	469	340	253	177	
13	693	642	503	371	278	194	
14	705	663	535	402	303	212	
15	716	682	565	432	328	230	
16	725	698	591	460	353	249	
17	732	711	616	487	377	267	
18	736	721	638	513	401	286	
19	743	729	658	538	425	304	
20	754	734	676	561	447	323	
21	767	738	692	583	470	341	
22	780	744	706	604	491	360	
23	790	754	717	623	512	378	
24	799	767	726	641	532	396	
25	807	780	732	658	551	414	
26	813	792	735	674	570	431	
27	820	803	740	688	588	449	
28	826	813	746	701	604	466	
29	831	821	756	712	621	482	
30	837	828	767	721	636	498	
31	842	835	780	728	651	514	
32	847	841	793	733	665	530	
33	852	846	805	736	678	545	
34	856	851	816	740	690	559	
35	861	856	827	745	701	573	
36	865	861	836	753	711	587	
37	870	866	844	763	719	601	
38	874	870	852	774	726	614	
39	878	874	859	786	731	626	
40	882	878	865	798	734	638	
41	885	882	871	810	737	650	
42	889	886	876	822	740	661	
43	893	890	881	832	745	672	
44	896	893	885	842	752	683	
45	900	897	890	852	761	692	

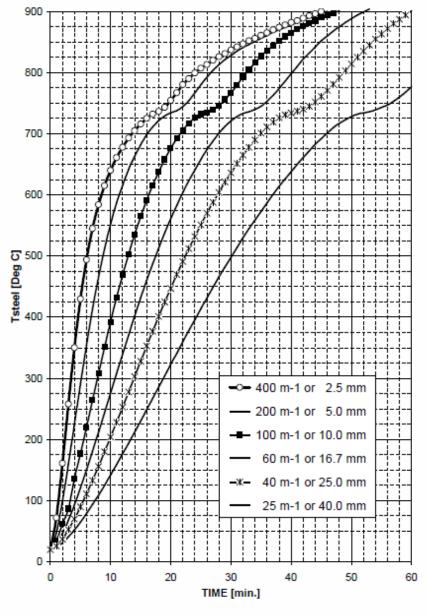
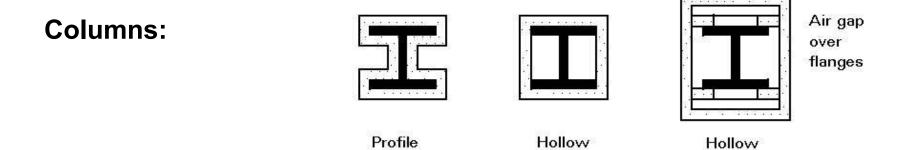


Figure I-3: temperatures as a function of time for various section factors

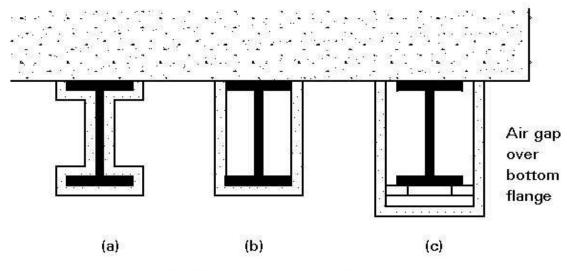
Passive structural fire protection

- Composite steel and concrete structures
 - In the past lining by bricks, embedding in concrete
 - Today preferable with structural function as well
- Insulating Boards
 - Gypsum, Mineral fiber, Vermiculite
 - + Easy to apply, aesthetically acceptable
 - Difficulties with complex details
- Cementations Sprays
 - Mineral fiber or vermiculite in cement binder
 - + Economy
 - Esthetics; normally used behind suspended ceilings.
- Intumescent Paints
 - Expands on heating to produce insulating layer
 - + Decorative finish under normal conditions
 - + May be done off-site
 - Durability

Exposed from three/four sides

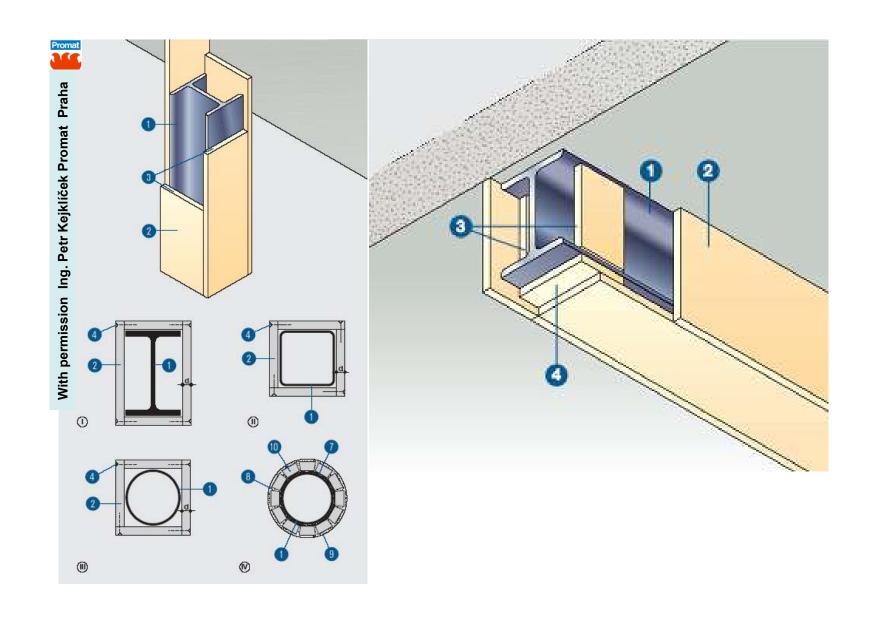


Beams:



- (a) Spray or intumescent
- (b) Board
- (c) Board

Insulating Boards













Cementations Sprays











Intumescent Paints















4.2.5.2 Internal steelwork insulated by fire protection material

(1) For a uniform temperature distribution in a cross-section, the temperature increase $\Delta \theta_{a,t}$ of an insulated steel member during a time interval Δt should be obtained from:

$$\Delta \theta_{\text{a},\text{t}} = \frac{\lambda_{\text{p}} A_{\text{p}} / V}{d_{\text{p}} c_{\text{a}} \rho_{\text{a}}} \frac{(\theta_{\text{g},\text{t}} - \theta_{\text{a},\text{t}})}{(1 + \phi/3)} \Delta t - (e^{\phi/10} - 1) \Delta \theta_{\text{g},\text{t}} \qquad (\text{but } \Delta \theta_{\text{a},\text{t}} \ge 0 \text{ if } \Delta \theta_{\text{g},\text{t}} \ge 0)$$

$$(4.27)$$

with:

$$\phi = \frac{c_{\rm p} \rho_{\rm p}}{c_{\rm a} \rho_{\rm a}} d_{\rm p} A_{\rm p} / V$$

where:

 A_p/V is the section factor for steel members insulated by fire protection material;

 A_p is the appropriate area of fire protection material per unit length of the member [m²];

V is the volume of the member per unit length [m³];

c_a is the temperature dependant specific heat of steel, from section 3 [J/kgK];

 c_p is the temperature independent specific heat of the fire protection material [J/kgK];

 d_p is the thickness of the fire protection material [m];

 Δt is the time interval [seconds];

 $\theta_{a,t}$ is the steel temperature at time $t[^{\circ}C]$;

 $\theta_{g,t}$ is the ambient gas temperature at time $t[^{\circ}C]$;

 $\Delta \theta_{g,t}$ is the increase of the ambient gas temperature during the time interval $\Delta t[K]$;

 $\lambda_{\rm p}$ is the thermal conductivity of the fire protection system [W/mK];

 ρ_a is the unit mass of steel, from section 3 [kg/m³];

 $\rho_{\rm p}$ is the unit mass of the fire protection material [kg/m³].

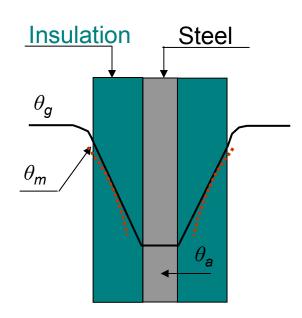
- (2) The values of c_p , λ_p and ρ_p should be determined as specified in section 3.
- (3) The value of Δt should not be taken as more than 30 seconds.
- (4) The area A_p of the fire protection material should generally be taken as the area of its inner surface, but for hollow encasement with a clearance around the steel member the same value as for hollow encasement without a clearance may be adopted.

Principle for equation 4.27:

The surface temperature of fire protection is considered to be the same as the gas temperature (no shadow effect)

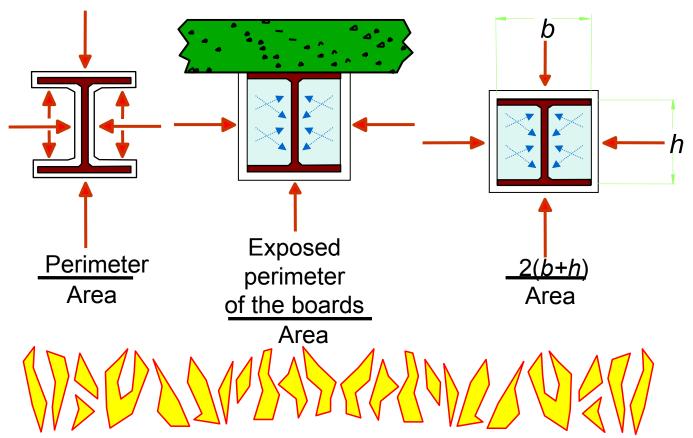
$$\theta g - \theta m << \theta m - \theta a$$

The heat transfer is performed through the fire protected layer.



Section factor A_n/V

for the fire protected steel element



The appropriate area of fire protection material per unit length of the member divided by the volume of the member per unit length.

Table 4.3: Section factor A_p/V for steel members insulated by fire protection material

Description	Section factor (A_p/V)	
Contour encasement of uniform thickness	steel perimeter steel cross-section area	
Hollow encasement of uniform thickness)	steel cross-section area	
Contour encasement of uniform thickness, exposed to fire on three sides	steel perimeter - b	
Hollow encasement of uniform thickness, exposed to fire on three sides) ^t	2h + b steel cross-section area	
	Contour encasement of uniform thickness) Hollow encasement of uniform thickness) Contour encasement of uniform thickness, exposed to fire on three sides Hollow encasement of uniform thickness, exposed to fire on three sides)	

If the specific heat of the protection material cp is conservatively neglected, Eq. 4.27 reduces to

$$\Delta \theta_{a,t} = \frac{\lambda_p A_p}{d_p V} \frac{(\theta_{g,t} - \theta_{a,t})}{c_a \rho_a} \Delta t$$

All parameters that define the steel section and the protection may be grouped into one single factor :

$$k_p = \frac{\lambda_p}{d_p} \frac{A_p}{V}$$

For a defined fire, it is quite convenient to perform the integration of the equation above once for various values of this factor *kp* and to build design aids in the form of tables or graphs.

Tables and graphs – protected steel

http://www.ct.upt.ro/users/RaulZaharia/Introduction fire design.pdf

Table I-2: temperature in protected sections subjected to the ISO fire

k _p [W/m ³ K]	200	400	600	800	1200	2000
Time [min.]	Steel temperature in °C					
0	20	20	20	20	20	20
10	37	54	70	85	113	163
20	60	97	130	160	215	304
30	84	139	188	232	306	421
40	108	181	244	298	388	514
50	132	222	296	359	459	589
60	156	260	345	414	520	650
70	179	298	391	465	573	699
80	202	333	433	510	620	730
90	225	367	472	552	661	743
100	247	399	509	589	695	773
110	268	430	542	623	721	816
120	289	459	573	654	734	859
130	310	486	602	681	744	900
140	330	512	629	705	765	935
150	349	537	654	723	795	965
160	368	560	677	733	828	990
170	386	582	697	739	861	1013
180	404	603	714	751	892	1032
190	422	623	727	769	921	1049
200	439	642	734	792	948	1065
210	455	660	738	817	972	1078
220	471	677	747	843	993	1090
230	487	692	760	869	1013	1101
240	502	706	777	893	1031	1112

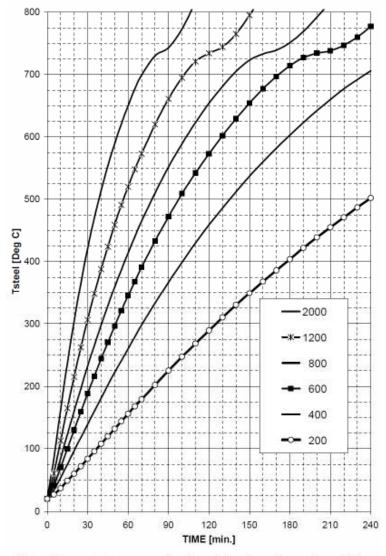
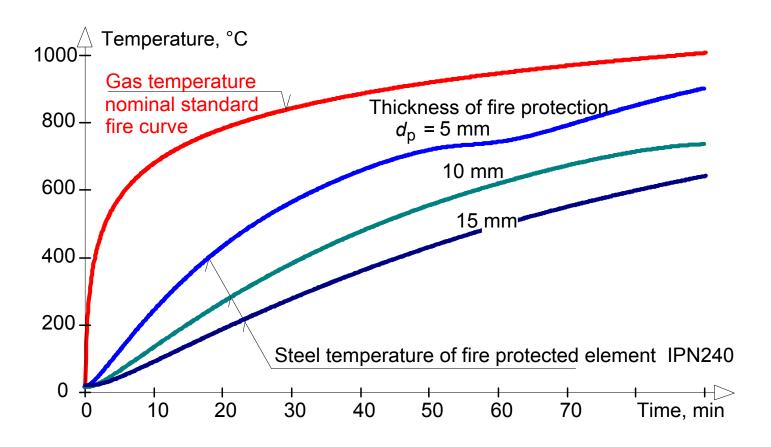


Figure I-5 : temperatures as a function of time for various values of the factor $k_{\rm p}$

Result of step by step procedure

Temperature of the fire protected steel element heated by nominal standard curve



Caution!

A common error that must be avoided is to consider the values of the thermal properties for the fire protection derived at ambient temperature, typically for applications such as thermal insulation in buildings.

This would lead to unsafe results in the fire situation, because the thermal conductivity has a tendency to increase with increasing temperature in most insulating materials.

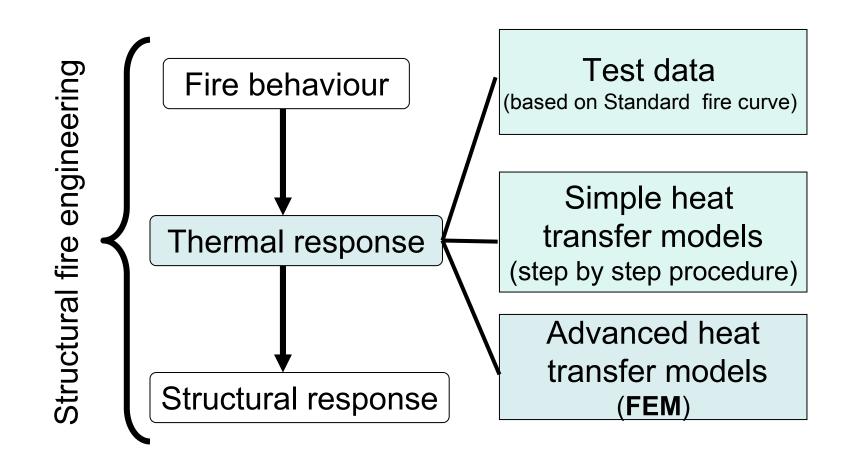
Material properties for preliminary design

Material	Unit mass ρ _n [kg/m³]	Thermal conductivity λ _n [W/(m.K)]	Specific heat c _n [J/(kg.K)]
SPRAYS	Pp[-9]	μ /2	p 2 (0 /2
- mineral fibre	300	0,12	1200
- vermiculite cement	350	0,12	1200
- perlite	350	0,12	1200
HIGH-DENSITY SPRAYS			
- vermiculite (or perlite) and cement	550	0,12	1100
- vermiculite (or perlite) and gypsum	650	0.12	1100
BOARDS			
- vermiculite (or perlite) and	800	0,2	1200
cement		0,2	
- fibre-silicate or fibre calcium-	600	0,15	1200
silicate		·	
- fibre-cement	800	0,15	1200
- gypsum board	800	0,20	1700
COMPRESSED FIBRE BOARDS			
- fibre-silicate, mineral- & stone- wool	150	0,2	1200
Concrete	2300	1,60	1000
Light weight concrete	1600	0,80	840
Concrete bricks	2200	1,00	1200
Bricks with holes	1000	0,40	1200
Solid bricks	2000	1,20	1200

Thermal conductivity models

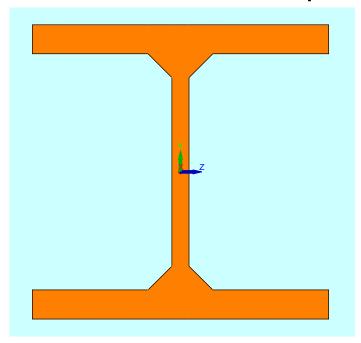
of common fire protection materials

Material	Density ρ, kg/m ³	Base value of specific heat, J/kg.K	Thermal conductivity, W/m.K
Rock fibre	155–180	900	$\lambda_{rock fibre} = 0.022 + 0.1475 \left(\frac{T}{1000}\right)^3$
Mineral wool	165	840	$\lambda_{mineral wool} = 0.03 + 0.2438 \left(\frac{T}{1000}\right)^3$
Calcium silicate	Various	900	$\lambda^* = \lambda_0^* + CT^3$
			$\lambda_0^* = 0.23 \frac{\rho}{1000}$ $C = 0.08 \times \frac{(2540 - \rho)}{2540}$
Vermiculite	Various	900	$\lambda^* = \lambda_0^* + CT^3$
			$\lambda_0^* = 0.27 \frac{\rho}{1000}$
			$C = 0.18 \times \frac{(1000 - \rho)}{1000}$

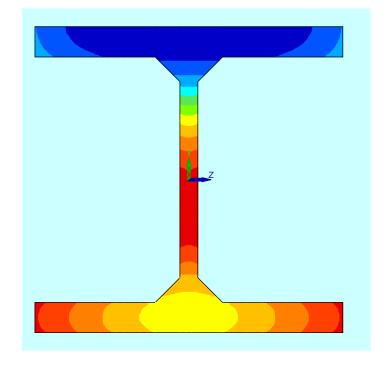


Thermal gradient on elements

Uniform temperature



Non uniform temperature



Temperatures development in structure during CTU edge bay test in Cardington













Recapitulation

- Describe the three models of heat transfer.
- Describe the major components of the net heat flux.
- What is influencing the emissivity during the fire?
- Which is the principle of the step by step procedure of heat transfer for the fire protected steel element?
- Which are the material thermal properties of fire protection that are taken into account in the evaluation of transfer of heat to the fire protected elements?
- Which is the difference in the calculation of the section factor for the fire protected and unprotected steel elements?



Thank you for your attention

Raul ZAHARIA raul.zaharia@upt.ro