## Suport de curs

## „Statica constructiilor - Structural analysis"

Inginerie civila
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An de studiu III

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## 1. Flexibility Method

Portal frame - 2 load cases


Load case 1

Degree of static indeterminacy

$$
\mathrm{d}=\mathrm{l}_{\mathrm{i}}+\mathrm{r}-3 \mathrm{e}
$$

$l_{i}$-number of internal links
$r$ - number of external links (reactions)
$e$ - number of elements
$d=9+4-3 \cdot 4=1 \quad$ (considering 4 elements rigidly connected, with 3 internal links between elements)
or
$d=0+4-3 \cdot 1=1$ (the 4 elements rigidly connected may be seen as a single body, therefore there are no internal links)

Primary structure


Compatibility equation

$$
\mathrm{D}_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{D}_{1 \mathrm{P}}^{0}=0
$$

$d_{11}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by the unit value of $X_{1}$, acting on the primary structure;
$d_{11} \cdot X_{1}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by $X_{1}$, acting on the primary structure;
$D_{1 P}^{0}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by the external loads, acting on the primary structure;
$D_{1 P}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by $X_{1}$ and by the external loads, acting on the primary structure; it must be zero, , because in this point and on the direction of X 1 , on the real structure exists a link which does not allow this displacement (pinned support).

Bending moment on the primary structure due to external loads

$$
\begin{array}{ll}
\left(\sum H\right)=0 & R_{H 1}^{P}=15 \mathrm{kN} \\
\left(\sum M\right)_{1}=0 & 50 \cdot 6+15 \cdot 6-R_{V 5}^{P} \cdot 12=0
\end{array}
$$

(Clockwise is considered positive)

$$
\Rightarrow \mathrm{R}_{\mathrm{V} 5}^{\mathrm{P}}=32.5 \mathrm{kN}
$$

$$
\left(\sum \mathrm{M}\right)_{5}=0 \quad \mathrm{R}_{\mathrm{V} 1}^{\mathrm{P}} \cdot 12-50 \cdot 6+15 \cdot 6=0 \quad \mathrm{R}_{\mathrm{V} 1}^{\mathrm{P}}=17.5 \mathrm{kN}
$$

Verification

$$
\left(\sum \mathrm{V}\right)=0 \quad \mathrm{R}_{\mathrm{V} 1}^{\mathrm{P}}+\mathrm{R}_{\mathrm{V} 5}^{\mathrm{P}}-50 \mathrm{kN}=0
$$



Bending moment on the primary structure due to $X_{1}=1$

$$
\begin{aligned}
& \left(\sum \mathrm{H}\right)=0 \quad \mathrm{R}_{\mathrm{H} 1}^{1}=1 \\
& \left(\sum \mathrm{M}\right)_{1}=0 \quad \mathrm{R}_{\mathrm{V} 1}^{1}=0 \\
& \left(\sum \mathrm{M}\right)_{5}=0 \quad \mathrm{R}_{\mathrm{V} 5}^{1}=0 \\
& \mathrm{EI} \cdot \mathrm{~d}_{11}=2\left[\frac{1}{1} \cdot \frac{6 \cdot(-6)}{2} \cdot \frac{2}{3} \cdot(-6)+\frac{1}{2} \cdot \frac{6.083 \cdot(-6)}{2} \cdot\left(\frac{2}{3} \cdot(-6)+\frac{1}{3} \cdot(-7)\right)+\frac{1}{2}\right. \\
& \\
& \left.\quad \cdot \frac{6.083 \cdot(-7)}{2} \cdot\left(\frac{2}{3} \cdot(-7)+\frac{1}{3} \cdot(-6)\right)\right]
\end{aligned}
$$

$\mathrm{EI} \cdot \mathrm{d}_{11}=401.514$
$\mathrm{EI} \cdot \mathrm{D}_{1 \mathrm{P}}^{0}=\frac{1}{1} \cdot \frac{6 \cdot 90}{2} \cdot \frac{2}{3} \cdot(-6)+\frac{1}{2} \cdot \frac{6.083 \cdot 90}{2} \cdot\left(\frac{2}{3} \cdot(-6)+\frac{1}{3} \cdot(-7)\right)+$ $+\frac{1}{2} \cdot \frac{6.083 \cdot 210}{2} \cdot\left(\frac{2}{3} \cdot(-7)+\frac{1}{3} \cdot(-6)\right)+\frac{1}{2} \cdot \frac{6.083 \cdot 210}{2} \cdot\left(\frac{2}{3} \cdot(-7)+\frac{1}{3} \cdot(-6)\right)$
EI $\cdot D_{1 P}^{0}=-6204.93$
$\mathrm{X}_{1}=-\frac{\mathrm{D}_{1 \mathrm{P}}^{0}}{\mathrm{~d}_{11}}=15.454 \mathrm{kN}$


Degree of static indeterminacy (the frame may be considered as having 4 elements rigidly connected each-other by 3 internal links + the tie, hinged to the frame, thus resulting 2 internal links on each end of the tie):

$$
\mathrm{d}=\mathrm{l}_{\mathrm{i}}+\mathrm{r}-3 \mathrm{e}=13+4-3 \cdot 5=2
$$

Primary structure


Compatibility equations

$$
\left\{\begin{array}{l}
D_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{d}_{12} \cdot \mathrm{X}_{2}+\mathrm{D}_{1 \mathrm{P}}^{0}=0 \\
D_{2 \mathrm{P}}=\mathrm{d}_{21} \cdot \mathrm{X}_{1}+\mathrm{d}_{22} \cdot \mathrm{X}_{2}+\mathrm{D}_{2 \mathrm{P}}^{0}=-\Delta \mathrm{l}_{2}
\end{array}\right.
$$

$d_{11}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by the unit value of $X_{1}$, acting on the primary structure;
$d_{22}$ - linear displacement at the point and in the direction of the redundant axial force $X_{2}$, caused by the unit value of $X_{2}$, acting on the primary structure;
$d_{11} \cdot X_{1}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by $X_{1}$, acting on the primary structure;
$d_{22} \cdot X_{2}$ - linear displacement at the point and in the direction of the redundant axial force $X_{2}$, caused by $X_{2}$, acting on the primary structure;
$d_{12}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by the unit value of $X_{2}$, acting on the primary structure;
$d_{21}$ - linear displacement at the point and in the direction of the redundant axial force $X_{2}$, caused by the unit value of $X_{1}$, acting on the primary structure;
$d_{12} \cdot X_{2}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by $X_{2}$, acting on the primary structure
$d_{21} \cdot X_{1}$ - linear displacement at the point and in the direction of the redundant axial force $\mathrm{X}_{2}$, caused by $\mathrm{X}_{1}$, acting on the primary structure;
$D_{1 P}^{0}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $\mathrm{X}_{1}$, caused by the external loads, acting on the primary structure
$D_{2 P}^{0}$ - linear displacement at the point and in the direction of the redundant axial force $X_{2}$, caused by the external loads, acting on the primary structure;
$D_{1 P}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by $X_{1}, X_{2}$ and by the external loads, acting on the primary structure; it must be zero, because in this point and on the direction of $X_{1}$, on the real structure exists a link which does not allow this displacement;
$D_{2 P}$ - linear displacement at the point and in the direction of the redundant axial force $X_{2}$, caused by $X_{1}, X_{2}$ and by the external loads, acting on the primary structure. This displacement is equal to the relative displacement between points 2 and $4\left(\Delta l_{2}\right)$ on the statically indeterminate structure. The "-" sign indicates that the displacement produced by the axial force in the tie $\left(\mathrm{X}_{2}\right)$ on the statically indeterminate structure, is equal and opposite to the displacement from the same force $X_{2}$ acting on the primary structure.

$$
\Delta l_{2}=\frac{l_{24}}{E_{t} \cdot A_{t}} \cdot X_{2}
$$

where:

$$
l_{24} \text { - length of the tie }
$$

$E_{t} \cdot A_{t}$ - axial rigidity of the tie

Bending moment on the primary structure due to $X_{2}=1$


EI $\cdot \mathrm{d}_{11}=2\left[\frac{1}{1} \cdot \frac{6 \cdot(-6)}{2} \cdot \frac{2}{3} \cdot(-6)+\frac{1}{2} \cdot \frac{6.083 \cdot(-6)}{2} \cdot\left(\frac{2}{3} \cdot(-6)+\frac{1}{3} \cdot(-7)\right)+\frac{1}{2}\right.$

$$
\left.\cdot \frac{6.083 \cdot(-7)}{2} \cdot\left(\frac{2}{3} \cdot(-7)+\frac{1}{3} \cdot(-6)\right)\right]
$$

$\mathrm{EI} \cdot \mathrm{d}_{11}=401.514$
$E I \cdot d_{12}=2\left[\frac{1}{2} \cdot \frac{6.083 \cdot(-6)}{2} \cdot\left(\frac{1}{3} \cdot(-1)\right)+\frac{1}{2} \cdot \frac{6.083 \cdot(-7)}{2} \cdot\left(\frac{2}{3} \cdot(-1)\right)\right]$

$$
=20.277
$$

EI $\cdot \mathrm{d}_{22}=2\left[\frac{1}{2} \cdot \frac{6.083 \cdot(-1)}{2} \cdot\left(\frac{2}{3} \cdot(-1)\right)\right]=2.03$
$\mathrm{EI} \cdot \mathrm{D}_{1 \mathrm{P}}^{0}=\frac{1}{1} \cdot \frac{6 \cdot 90}{2} \cdot \frac{2}{3} \cdot(-6)+\frac{1}{2} \cdot \frac{6.083 \cdot 90}{2} \cdot\left(\frac{2}{3} \cdot(-6)+\frac{1}{3} \cdot(-7)\right)+\frac{1}{2} \cdot \frac{6.083 \cdot 210}{2}$

$$
\cdot\left(\frac{2}{3} \cdot(-7)+\frac{1}{3} \cdot(-6)\right)+\frac{1}{2} \cdot \frac{6.083 \cdot 210}{2} \cdot\left(\frac{2}{3} \cdot(-7)+\frac{1}{3} \cdot(-6)\right)
$$

$\mathrm{EI} \cdot \mathrm{D}_{1 \mathrm{P}}^{0}=-6204.93$
$\mathrm{EI} \cdot \mathrm{D}_{2 \mathrm{P}}^{0}=\frac{1}{2} \cdot \frac{6.083 \cdot 90}{2} \cdot\left(\frac{1}{3} \cdot(-1)\right)+\frac{1}{2} \cdot \frac{6.083 \cdot 210}{2} \cdot\left(\frac{2}{3} \cdot(-1)\right)+\frac{1}{2}$

$$
\cdot \frac{6.083 \cdot 210}{2} \cdot\left(\frac{2}{3} \cdot(-1)\right)
$$

$\mathrm{EI} \cdot \mathrm{D}_{2 \mathrm{P}}^{0}=-471.433$

$$
\begin{aligned}
& \left\{\begin{array}{l}
\mathrm{D}_{1 \mathrm{P}}=\frac{401.514}{\mathrm{EI}} \cdot \mathrm{X}_{1}+\frac{20.277}{\mathrm{EI}} \cdot \mathrm{X}_{2}-\frac{6204.93}{\mathrm{EI}}=0 \\
\mathrm{D}_{2 \mathrm{P}}=\frac{20.277}{\mathrm{EI}} \cdot \mathrm{X}_{1}+\frac{2.03}{\mathrm{EI}} \cdot \mathrm{X}_{2}-\frac{471.433}{\mathrm{EI}}=-\frac{12.0}{\mathrm{E}_{\mathrm{t}} \mathrm{~A}_{\mathrm{t}}} \cdot \mathrm{X}_{2}
\end{array}\right. \\
& \left\{\begin{array}{l}
\mathrm{D}_{1 \mathrm{P}}=401.514 \cdot \mathrm{X}_{1}+20.277 \cdot \mathrm{X}_{2}-6204.93=0 \\
\mathrm{D}_{2 \mathrm{P}}=20.277 \cdot \mathrm{X}_{1}+2.03 \cdot \mathrm{X}_{2}-471.433=-12 \frac{\mathrm{EI}}{\mathrm{E}_{\mathrm{t}} \mathrm{~A}_{\mathrm{t}}} \cdot \mathrm{X}_{2} \\
\\
\quad \frac{E I}{E_{t} A_{t}}=\frac{1.512 \cdot 10^{5} \mathrm{kNm}}{\mathrm{E}_{t} \cdot \mathrm{~A}_{t}=1.4845 \cdot 10^{5} \mathrm{kN}}=1.0165 \mathrm{~m}^{2}
\end{array}\right. \\
& X_{1}=14.852 \mathrm{kN} \\
& X_{2}=11.95 \mathrm{kN}
\end{aligned}
$$

Portal frame with inclined elements


Degree of static indeterminacy (the structure may be seen as two bodies connected by a hinge, thus having 2 internal links):

$$
d=l_{i}+r-3 e=2+5-3 \cdot 2=1
$$

- In the first case, the unknown is the moment reaction in point 5:


Bending moment on the primary structure due to external loads


## Compatibility equation

$$
\mathrm{D}_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{D}_{1 \mathrm{P}}^{0}=0
$$

The terms in the compatibility equation represent:
$d_{11}$ - rotation at the point and in the direction of the redundant moment reaction X 1 , caused by the unit value of X 1 , acting on the primary structure
$D_{1 P}^{0}$ - rotation at the point and in the direction of the redundant moment reaction X 1 , caused by the external loads, acting on the primary structure
$D_{1 P}$ - rotation at the point and in the direction of the redundant moment reaction X 1 , caused by X1 and by the external loads, acting on the primary structure; it must be zero, because in this point and on the direction of X 1 , on the real structure exists a link which does not allow this rotation (fixed support).

The expression for bending moment between nodes 3 and 4 is:

$$
\left.\begin{array}{l}
\begin{array}{l}
M_{P}^{0}=R_{V 1}^{P} \cdot x-p \cdot(x+2) \cdot\left(\frac{x+2}{2}\right)-R_{H 1}^{0} \cdot 5 \\
\quad=87.5 \cdot x-7.5 \cdot\left(x^{2}+4 x+4\right)-70 \\
\quad=-7.5 \cdot x^{2}+57.5 \cdot x-100
\end{array} \\
\begin{array}{rl}
m_{1}=R_{V 1}^{1} \cdot x-R_{H 1}^{1} \cdot 5=0.111 \cdot x-0,555
\end{array} \\
\begin{array}{l}
d_{11}=\int \frac{m_{1} \cdot m_{1}}{E I} d x
\end{array} \\
E I \cdot d_{11}=\frac{1}{1} \cdot \frac{0.555 \cdot 5}{2} \cdot 0.37+\frac{1}{2} \cdot \frac{0.555 \cdot 5}{2} \cdot 0.37+\frac{1}{1} \cdot \frac{1 \cdot 6.403}{2} \cdot 0.667=2,904
\end{array}\right] \begin{aligned}
& D_{1 P}^{0}=\int \frac{m_{1} \cdot M_{p}^{0}}{E I} d x
\end{aligned}
$$

$\mathrm{EI} \cdot \mathrm{D}_{1 \mathrm{P}}^{0}=\frac{1}{1} \cdot \frac{-70 \cdot 5}{2} \cdot \frac{2}{3} \cdot(-0,555)+\frac{1}{2}$

$$
\int_{0}^{5}(0.111 \cdot x-0,555) \cdot\left(-7.5 \cdot x^{2}+57.5 \cdot x-100\right) d x
$$

$\mathrm{EI} \cdot \mathrm{D}_{1 \mathrm{P}}^{0}=64.75+\frac{1}{2} \cdot \int_{0}^{5}\left(-0.8325 \cdot \mathrm{x}^{3}+6.3825 \cdot \mathrm{x}^{2}-11.1 \cdot \mathrm{x}+\right.$

$$
\left.+4.162 \cdot x^{2}-31.9125 \cdot x+55.5\right) \mathrm{dx}
$$

$E I \cdot D_{1 P}^{0}=64.75+\frac{1}{2} \cdot \int_{0}^{5}\left(-0.8325 \cdot x^{3}+10.545 \cdot x^{2}-43.0125 \cdot x+55.5\right) d x$
$\mathrm{EI} \cdot \mathrm{D}_{1 \mathrm{P}}^{0}=64.75+\left.\frac{1}{2} \cdot\left(-0.8325 \cdot \frac{\mathrm{x}^{4}}{4}+10.545 \cdot \frac{\mathrm{x}^{3}}{3}-43.0125 \cdot \frac{\mathrm{x}^{2}}{2}+55.5 \cdot \mathrm{x}\right)\right|_{0} ^{5}$
$E I \cdot D_{1 P}^{0}=64.75+24.57=89.32$
$\Rightarrow X_{1}=-\frac{D_{1 \mathrm{P}}^{0}}{d_{11}}=-30.75 \mathrm{kNm}$

Symmetric portal frame


Degree of static indeterminacy

$$
d=l_{i}+r-3 e=6+6-3 \cdot 3=3
$$

The structure is symmetric. The load may be decomposed in two load cases, which will be treated separately: one symmetric and one antisymmetric.


The deformed shapes of the structure for the two load cases are symmetric and antisymmetric, respectively. For each load case, in the point situated in the middle of the beam (in the axis of symmetry of the structure), some displacements are zero:


In order to simplify the calculation, half of structure may be considered for both load cases. For the point in the middle of the beam, external links corresponding to the zero displacements from figures above may be considered, by means of appropriate supports:


Consequently, the following primary structures may be considered for the two structures:


The symmetric structure
Bending moment on the primary structure due to external loads


Bending moment on the primary structure due to $X_{1}=1$


Bending moment on the primary structure due to $X_{2}=1$


Compatibility equations

$$
\left\{\begin{array}{l}
\mathrm{D}_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{d}_{12} \cdot \mathrm{X}_{2}+\mathrm{D}_{1 \mathrm{P}}^{0}=0 \\
\mathrm{D}_{2 \mathrm{P}}=\mathrm{d}_{21} \cdot \mathrm{X}_{1}+\mathrm{d}_{22} \cdot \mathrm{X}_{2}+\mathrm{D}_{2 \mathrm{P}}^{0}=0
\end{array}\right.
$$

EI $\cdot \mathrm{d}_{11}=\frac{1}{1} \cdot \frac{4 \cdot(-4)}{2} \cdot \frac{2}{3} \cdot(-4)=21.333$
$\mathrm{EI} \cdot \mathrm{d}_{12}=\frac{1}{1} \cdot \frac{4 \cdot(-4)}{2} \cdot 1=-8$
EI $\cdot \mathrm{d}_{22}=\frac{1}{1} \cdot(4 \cdot 1) \cdot 1+\frac{1}{4.612} \cdot(4.5 \cdot 1) \cdot 1=4.9757$
$\mathrm{EI} \cdot \mathrm{D}_{1 \mathrm{P}}^{0}=\frac{1}{1} \cdot[4 \cdot(-121.5)] \cdot \frac{1}{2} \cdot(-4)=972$
$\mathrm{EI} \cdot \mathrm{D}_{2 \mathrm{P}}^{0}=\frac{1}{1} \cdot[4 \cdot(-121.5)] \cdot 1+\frac{1}{4.612} \cdot\left[\frac{1}{3} \cdot 4.5 \cdot(-121.5)\right] \cdot 1=-525.516$
$\mathrm{X}_{1}=-15.001 \mathrm{kN}$
$\mathrm{X}_{2}=81.497 \mathrm{kNm}$

## The antisymmetric structure

Bending moment on the primary structure due to external loads



Bending moment on the primary structure due to $X_{1}=1$


Compatibility equations

$$
\begin{aligned}
& \quad \mathrm{D}_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{D}_{1 \mathrm{P}}^{0}=0 \\
& \mathrm{EI} \cdot \mathrm{~d}_{11}=\frac{1}{1} \cdot[4 \cdot(-4.5)] \cdot(-4.5)+\frac{1}{4.612} \cdot \frac{4.5 \cdot(-4.5)}{2} \cdot \frac{2}{3} \cdot(-4.5)=87.586 \\
& \mathrm{EI} \cdot \mathrm{D}_{1 \mathrm{P}}^{0}=\frac{1}{1} \cdot\left[\frac{1}{3} \cdot 4 \cdot(-28)\right] \cdot(-4.5)=168 \\
& \mathrm{X}_{1}=-1.918 \mathrm{kN}
\end{aligned}
$$

## 2. TRUSSES

## Problem 1



Degree of static indeterminacy
$\mathrm{d}=\mathrm{b}+\mathrm{r}-2 \cdot \mathrm{j}=10+3-2 \cdot 6=1$
b - number of bars
$r$ - number of external links (reactions)
$j$ - number of joints
Primary structure


Compatibility equation

$$
\mathrm{D}_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{D}_{1 \mathrm{P}}^{0}=-\Delta \mathrm{l}_{1}
$$

$d_{11}$ - linear displacement at the point and in the direction of the redundant axial force $X_{1}$, caused by the unit value of $X_{1}$, acting on the primary structure;
$D_{1 P}^{0}$ - linear displacement at the point and in the direction of the redundant axial force $\mathrm{X}_{1}$, caused by the external loads, acting on the primary structure;
$D_{1 P}$ - displacement at the point and in the direction of the redundant axial force $X_{1}$, caused by $X_{1}$ and by the external loads, acting on the primary structure;
$\Delta l_{1}$ - relative displacement of joints 2 and 3 on the real structure. The "-" sign indicates that the displacement produced by the axial force $\mathrm{X}_{1}$ on the statically indeterminate structure is equal and opposite to the displacement from the same force acting on the primary structure.

$$
\Delta l_{1}=\frac{l_{23}}{\mathrm{EA}} \cdot X_{1}
$$

where:

$$
\begin{aligned}
& l_{23}-\text { length of the bar 2-3 } \\
& \mathrm{EA}_{23} \text { - axial rigidity of the bar 2-3 }
\end{aligned}
$$

## Problem 2



## Degree of static indeterminacy

$$
d=b+r-2 \cdot j=13+4-2 \cdot 8=1
$$

## Primary structure



Compatibility equation

$$
\mathrm{D}_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{D}_{1 \mathrm{P}}^{0}=0
$$

$d_{11}$ - linear displacement at the point and in the direction of the redundant vertical reaction $X_{1}$, caused by the unit value of $X_{1}$, acting on the primary structure;
$D_{1 P}^{0}$ - linear displacement at the point and in the direction of the redundant vertical reaction $X_{1}$, caused by the external loads, acting on the primary structure;
$D_{1 P}$ - displacement at the point and in the direction of the redundant vertical reaction $X_{1}$, caused by $X_{1}$ and by the external loads, acting on the primary structure; it must be zero, because in this point, on the real structure exists a link which prevents this vertical displacement (roller support).

## Problem



$$
d=b+r-2 \cdot j=14+4-2 \cdot 8=2
$$

Primary structure


Compatibility equations

$$
\left\{\begin{array}{l}
\mathrm{D}_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{d}_{12} \cdot \mathrm{X}_{2}+\mathrm{D}_{1 \mathrm{P}}^{0}=0 \\
\mathrm{D}_{2 \mathrm{P}}=\mathrm{d}_{21} \cdot \mathrm{X}_{1}+\mathrm{d}_{22} \cdot \mathrm{X}_{2}+\mathrm{D}_{2 \mathrm{P}}^{0}=-\Delta \mathrm{l}_{2}
\end{array}\right.
$$

## Problem



Degree of static indeterminacy

$$
d=b+r-2 \cdot j=8+4-2 \cdot 5=2
$$

Primary structure


Compatibility equations

$$
\left\{\begin{array}{l}
D_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{d}_{12} \cdot \mathrm{X}_{2}+\mathrm{D}_{1 \mathrm{P}}^{0}=0 \\
D_{2 \mathrm{P}}=\mathrm{d}_{21} \cdot \mathrm{X}_{1}+\mathrm{d}_{22} \cdot \mathrm{X}_{2}+\mathrm{D}_{2 \mathrm{P}}^{0}=-\Delta \mathrm{l}_{2}
\end{array}\right.
$$

## ARCHES

## Problem

Circular arch with constant cross-section

$$
\frac{f}{L}=\frac{1}{6}<\frac{1}{5}
$$



## Cross-section


Degree of static indeterminacy

$$
d=l_{i}+r-3 e=0+4-3 \cdot 1=1
$$

Primary structure


Compatibility equation

$$
\mathrm{D}_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{D}_{1 \mathrm{P}}^{0}=0
$$

$d_{11}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by the unit value of $X_{1}$, acting on the primary structure;
$D_{1 P}^{0}$ - linear displacement at the point and in the direction of the redundant horizontal reaction $X_{1}$, caused by the external loads, acting on the primary structure;
$D_{1 P}$ - displacement at the point and in the direction of the redundant horizontal reaction $\mathrm{X}_{1}$, caused by $\mathrm{X}_{1}$ and by the external loads, acting on the primary structure; it must be zero, because in this point and on the direction of $X_{1}$, on the real structure exists a link which does not allow this displacement (pinned support).

## Problem

Parabolic arch with the same span, rise, cross-section and external load as for the structure from problem 1.9.

$$
\mathrm{y}(\mathrm{x})=\frac{4 \cdot f \cdot x}{l^{2}} \cdot(l-x)
$$



Primary structure


Compatibility equation

$$
D_{1 \mathrm{P}}=d_{11} \cdot X_{1}+D_{1 P}^{0}=0
$$

## Problem

Tied parabolic arch with the same span, rise, cross-section and load as for the arch from Problem 1.10.

$$
\begin{aligned}
& E=2.1 \cdot 10^{4} \frac{\mathrm{~N}}{\mathrm{~mm}^{2}}, \quad E_{t}=2.1 \cdot 10^{5} \frac{\mathrm{~N}}{\mathrm{~mm}^{2}}, \quad A_{t}=6362 \mathrm{~mm}^{2} \\
& y(x)=\frac{4 \cdot f \cdot x}{l^{2}} \cdot(l-x)
\end{aligned}
$$




Cross-section

$60 \mathrm{~cm} \mid$

Degree of static indeterminacy

$$
d=l_{i}+r-3 e=4+3-3 \cdot 2=1
$$

Primary structure


Compatibility equation

$$
\mathrm{D}_{1 \mathrm{P}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{D}_{1 \mathrm{P}}^{0}=-\Delta \mathrm{l}_{1}
$$

$d_{11}$ - linear displacement at the point and in the direction of the redundant axial force $X_{1}$, caused by the unit value of $X_{1}$, acting on the primary structure;
$D_{1 P}^{0}$ - displacement at the point and in the direction of the redundant axial force $X_{1}$, caused by the external loads, acting on the primary structure;
$D_{1 P}$ - displacement at the point and in the direction of the redundant axial force $\mathrm{X}_{1}$, caused by $\mathrm{X}_{1}$ and by the external loads, acting on the primary structure, equal to the relative horizontal displacement of the supports on the real statically indeterminate arch $\Delta l_{1}$. The "-" sign indicates that the displacement produced by the axial force in the tie ( $\mathrm{X}_{1}$ ) on the statically indeterminate arch, is equal and opposite to the displacement of the same force $\mathrm{X}_{1}$ acting on the primary structure.

## Displacement method

## Problem



Primary structure


Equilibrium equations

$$
\left\{\begin{array}{l}
R_{1 \mathrm{P}}=r_{11} \cdot D_{1}+r_{12} \cdot D_{2}+\mathrm{R}_{1 P}^{0}=0 \\
R_{2 P}=r_{21} \cdot D_{1}+r_{22} \cdot D_{2}+\mathrm{R}_{2 P}^{0}=0
\end{array}\right.
$$

$r_{11}$ - moment reaction in the rotational restraint 1 , caused by the unit rotation of $D_{1}$ imposed on the primary structure;
$r_{11} \cdot D_{1}$ - moment reaction in the rotational restraint 1 , caused by $D_{1}$ imposed on the primary structure;
$r_{22}$ - moment reaction in the rotational restraint 2 , caused by the unit rotation of $D_{2}$ imposed on the primary structure;
$r_{22} \cdot D_{2}$ - moment reaction in the rotational restraint 2 , caused by $D_{2}$ imposed on the primary structure;
$r_{12}$ - moment reaction in the rotational restraint 1, caused by the unit rotation of $D_{2}$ imposed on the primary structure;
$r_{12} \cdot D_{2}$ - moment reaction in the rotational restraint 1 , caused by $D_{2}$ imposed on the primary structure;
$r_{21}$ - moment reaction in the rotational restraint 2 , caused by the unit rotation of $D_{1}$ imposed on the primary structure;
$r_{21} \cdot D_{1}$ - moment reaction in the rotational restraint 2 , caused by $D_{1}$ imposed on the primary structure;
$\mathrm{R}_{1 P}^{0}$ - moment reaction in the rotational restraint 1 , caused by the external loads on the primary structure;
$\mathrm{R}_{2 P^{0}}^{0}$ - moment reaction in the rotational restraint 2, caused by the external loads on the primary structure;
$R_{1 \mathrm{P}}$ - moment reaction in the rotational restraint 1, caused by the rotations $D_{1}, D_{2}$ and the external loads on the primary structure; it must be zero, because on the real structure, joint (3) is free to rotate on the direction of $D_{1}$;
$R_{2 \mathrm{P}}$ - moment reaction in the rotational restraint 2, caused by the rotation $D_{1}, D_{2}$ and the external loads on the primary structure; it must be zero, because on the real structure joint (4) is free to rotate on the direction of $D_{2}$.

Fixed end moments due to external loads


The bending moment diagram between point 1 and 3 is the result of adding the bending moment due to the point load $(30 \mathrm{kN})$ and the bending moment due to the uniform distributed load as shown in the figure bellow:


Fixed end moments due to unit rotation $D_{1}=1$


Fixed end moments due to the unit rotation $D_{2}=1$


Moment reactions in the restrained joints

$R_{1 P}^{0}+18+15-48=0 \Rightarrow R_{1 P}^{0}=15 \mathrm{kNm}$
$\mathrm{r}_{11}=0.667 \cdot \mathrm{EI}+0.75 \cdot \mathrm{EI} \Rightarrow \mathrm{r}_{11}=1.417 \cdot \mathrm{EI}$
$\mathrm{R}_{2 \mathrm{P}}^{0}+60-18=0 \Rightarrow \mathrm{R}_{2 \mathrm{P}}^{0}=-42 \mathrm{kNm}$
$\mathrm{r}_{12}-0.333 \cdot \mathrm{EI}=0 \Rightarrow \mathrm{r}_{12}=0.333 \cdot \mathrm{EI}=\mathrm{r}_{21}$
$\mathrm{r}_{22}=1.417 \mathrm{EI}$
$\left\{1.417 \cdot \mathrm{EI} \cdot \mathrm{D}_{1}+0.333 \cdot \mathrm{EI} \cdot \mathrm{D}_{2}+15=0\right.$
$\left\{0.333 \cdot \mathrm{EI} \cdot \mathrm{D}_{1}+1.417 \cdot \mathrm{EI} \cdot \mathrm{D}_{2}-42=0\right.$

$$
\begin{aligned}
& \Rightarrow D_{1}=-\frac{18.576}{\mathrm{EI}} \\
& \Rightarrow D_{2}=\frac{34}{\mathrm{EI}}
\end{aligned}
$$

## Problem



Primary structure


Equilibrium equation

$$
\mathrm{R}_{1 \mathrm{P}}=\mathrm{r}_{11} \cdot \mathrm{D}_{1}+\mathrm{R}_{1 \mathrm{P}}^{0}=0
$$

Fixed end moments due to external loads


Fixed end moments due to unit rotation $D_{1}=1$


Moment reactions in the restrained joint 3

$$
\begin{aligned}
& 31.25-20 \\
& \mathrm{R}_{1 \mathrm{P}}^{0}+20-31.25=0 \Rightarrow \mathrm{R}_{1 \mathrm{P}}^{0}=11.25 \\
& \mathrm{r}_{11}-2.25 \mathrm{EI}-\mathrm{EI}-1.8 \mathrm{EI}=0 \Rightarrow \mathrm{r}_{11}=5.05 \mathrm{EI} \\
& 5.05 \mathrm{EI} \cdot \mathrm{D}_{1}+11.25=0 \\
& \mathrm{D}_{1}=-\frac{2,228}{\mathrm{EI}}
\end{aligned}
$$

## Problem

The structure is similar to the one from problem 1.13, but a rigid joint is considered in point 2.


Primary structure


Equilibrium equations

$$
\left\{\begin{array}{l}
R_{1 P}=r_{11} \cdot D_{1}+r_{12} \cdot D_{2}+R_{1 P}^{0}=0 \\
R_{2 P}=r_{21} \cdot D_{1}+r_{22} \cdot D_{2}+R_{2 P}^{0}=0
\end{array}\right.
$$

Fixed end moments due to external loads


Fixed end moments due to unit rotation $D_{1}=1$


Fixed end moments due to unit rotation $D_{2}=1$


Moment reactions in the restrained joints
(2)

(2) 2.4 El


$\mathrm{R}_{1 \mathrm{P}}^{0}+20.83-4=0 \Rightarrow \mathrm{R}_{1 \mathrm{P}}^{0}=-16.83 \mathrm{kNm}$
$\mathrm{R}_{2 \mathrm{P}}^{0}+20-20.83=0 \Rightarrow \mathrm{R}_{2 \mathrm{P}}^{0}=0.83 \mathrm{kNm}$
$\mathrm{r}_{11}-2.4 \mathrm{EI}-2 \mathrm{EI}=0 \Rightarrow \mathrm{r}_{11}=4.4 \mathrm{EI}$
$\mathrm{r}_{22}-2.25 \mathrm{EI}-\mathrm{EI}-2,4 \cdot \mathrm{EI}=0 \Rightarrow \mathrm{r}_{22}=5.65 \mathrm{EI}$
$r_{12}-1.2 \mathrm{EI}=0 \Rightarrow r_{12}=1.2 \mathrm{EI}=\mathrm{r}_{21}$
$\left\{4.4 \cdot D_{1}+1.2 \cdot D_{2}-16.83=0\right.$
$\left\{1.2 \cdot D_{1}+5.65 \cdot D_{2}+0.83=0\right.$

$$
\begin{aligned}
& \mathrm{D}_{1}=\frac{4.103}{\mathrm{EI}} \\
& \mathrm{D}_{2}=-\frac{1.018}{\mathrm{EI}}
\end{aligned}
$$

## Problem

The structure is similar to the one from problem 1.13, but a roller is considered in point 5 .


Primary structure


Equilibrium equations

$$
\left\{\begin{array}{l}
R_{1 \mathrm{P}}=\mathrm{r}_{11} \cdot \mathrm{D}_{1}+\mathrm{r}_{12} \cdot \mathrm{D}_{2}+\mathrm{R}_{1 \mathrm{P}}^{0}=0 \\
\mathrm{R}_{2 \mathrm{P}}=\mathrm{r}_{21} \cdot \mathrm{D}_{1}+\mathrm{r}_{22} \cdot \mathrm{D}_{2}+\mathrm{R}_{2 \mathrm{P}}^{0}=0
\end{array}\right.
$$

$r_{11}$ - moment reaction in the rotational restraint 1 , caused by the unit rotation $\mathrm{D}_{1}$ imposed on the primary structure;
$r_{22}$ - force reaction in the linear displacement restraint 2, caused by the unit linear displacement $\mathrm{D}_{2}$ imposed on the primary structure;
$r_{12}$ - moment reaction in the rotational restraint 1 , caused by the unit linear displacement $D_{2}$ imposed on the primary structure;
$r_{21}$ - force reaction in the linear displacement restraint 2, caused by the unit rotation $D_{1}$ imposed on the primary structure;
$\mathrm{R}_{1 P}^{0}$ - moment reaction in the rotational restraint 1 , caused by the external loads on the primary structure;
$\mathrm{R}_{2 P}^{0}$ - force reaction in the linear displacement restraint 2, caused by the external loads on the primary structure;
$R_{1 \mathrm{P}}$ - moment reaction in the rotational restraint 1 caused by the rotation $D_{1}$, the linear displacement $D_{2}$ and the external loads on the primary structure; it must be zero, because on the real structure joint (3) is free to rotate in the direction of $D_{1}$;
$R_{2 \mathrm{P}}$ - force reaction in the linear displacement restraint 2 , caused by the rotation $D_{1}$, the linear displacement $D_{2}$ and the external loads on the primary structure; it must be zero, because on the real structure the roller in point (5) allows the lateral displacement.

## Problem

The structure is similar to the one from problem1.15, but the beams 2-3 and $3-5$ are hinged to the column 3-4.


## Primary structure



Equilibrium equations

$$
\mathrm{R}_{1 \mathrm{P}}=\mathrm{r}_{11} \cdot \mathrm{D}_{1}+\mathrm{R}_{1 \mathrm{P}}^{0}=0
$$

$r_{11}$ - force reaction in the linear displacement restraint 1 , caused by the unit displacement $D_{1}$ imposed on the primary structure;
$\mathrm{R}_{1 P}^{0}$ - horizontal reaction in the linear displacement restraint 1 , caused by the external loads on the primary structure;
$R_{1 \mathrm{P}}$ - force reaction in the linear displacement restraint 1 , caused by the linear displacement $D_{1}$ and the external loads on the primary structure

Fixed end moments due to external loads

## Problem



Primary structure


Equilibrium equations

$$
\left\{\begin{array}{l}
R_{1 P}=r_{11} \cdot D_{1}+r_{12} \cdot D_{2}+R_{1 P}^{0}=0 \\
R_{2 P}=r_{21} \cdot D_{1}+r_{22} \cdot D_{2}+R_{2 P}^{0}=0
\end{array}\right.
$$

## Problem

The structure is similar to the one from problem 1.17, but a roller is considered in point 7. The degree of kinematic indeterminacy becomes higher than for the structure from problem 1.17, but the degree of static indeterminacy is lower. It may be observed that this structure is statically determinate, but it still may be analysed using the displacement method.


Primary structure


Equilibrium equations

$$
\left\{\begin{array}{l}
R_{1 \mathrm{P}}=\mathrm{r}_{11} \cdot \mathrm{D}_{1}+\mathrm{r}_{12} \cdot \mathrm{D}_{2}+\mathrm{r}_{13} \cdot \mathrm{D}_{3}+\mathrm{R}_{1 \mathrm{P}}^{0}=0 \\
\mathrm{R}_{2 \mathrm{P}}=\mathrm{r}_{21} \cdot \mathrm{D}_{1}+\mathrm{r}_{22} \cdot \mathrm{D}_{2}+\mathrm{r}_{23} \cdot \mathrm{D}_{3}+\mathrm{R}_{2 \mathrm{P}}^{0}=0 \\
\mathrm{R}_{3 \mathrm{P}}=\mathrm{r}_{31} \cdot \mathrm{D}_{1}+\mathrm{r}_{32} \cdot \mathrm{D}_{2}+\mathrm{r}_{33} \cdot \mathrm{D}_{3}+\mathrm{R}_{3 \mathrm{P}}^{0}=0
\end{array}\right.
$$

## Problem

The geometry of the two structures bellow is similar, excepting for the bending rigidity of column 1-2.


Primary structure (the same for both structures)


## Problem



Primary structure


Equilibrium equations

$$
\left\{\begin{array}{l}
R_{1 P}=r_{11} \cdot D_{1}+r_{12} \cdot D_{2}+R_{1 P}^{0}=0 \\
R_{2 P}=r_{21} \cdot D_{1}+r_{22} \cdot D_{2}+R_{2 P}^{0}=0
\end{array}\right.
$$

## 3. STATICALLY INDETERMINATE STRUCTURES SUBJECTED TO TEMPERATURE VARIATION

### 2.1 Flexibility Method

## Problem

The structure is the same as the one from problem 1.3


The degree of static indeterminacy of structure is 1 , while the degree of kinematic indeterminacy is 2 . Therefore, for this problem it is easier to use the flexibility method (1 unknown) instead of the displacement method (2 unknowns- one rotation and one linear displacement)

Primary structure


Compatibility equation

$$
\mathrm{D}_{1 \mathrm{t}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{D}_{1 \mathrm{t}}^{0}=0
$$

$d_{11}$ - rotation at the point and in the direction of the redundant moment reaction $X_{1}$, caused by the unit value of $X_{1}$, acting on the primary structure
$D_{1 t}^{0}$ - rotation at the point and in the direction of the redundant moment reaction $X_{1}$, caused by the temperature variation on the primary structure;
$D_{1 \mathrm{t}}$ - rotation at the point and in the direction of the redundant moment reaction $X_{1}$, caused by $X_{1}$ and by temperature variation on the primary structure; it must be zero, because in this point and on the direction of $X_{1}$, on the real structure exists a link which does not allow this displacement (fixed support)
Temperature diagrams


Bending moment and axial force due to unit rotation $X_{1}=1$


The value for $d_{11}$ is the same as for the structure from Problem 1.3

$$
\mathrm{d}_{11}=\frac{2.904}{\mathrm{EI}}
$$

$$
\begin{aligned}
& D_{1 t}^{0}=\sum \int \mathrm{n}_{1} \cdot \alpha \cdot \mathrm{t}_{\mathrm{m}} \mathrm{dx}+\int \mathrm{m}_{1} \cdot \frac{\alpha \cdot \Delta \mathrm{t}}{\mathrm{~h}} \mathrm{dx} \\
& =\alpha \cdot(-0.111 \cdot 5 \cdot 10-0.111 \cdot 5 \cdot 5+0.01732 \cdot 6.403 \cdot 5)+ \\
& +\alpha \cdot\left(\frac{20}{0.4} \cdot \frac{1}{2} \cdot(-0.555) \cdot 5+\frac{30}{0.5} \cdot \frac{1}{2} \cdot(-0.555) \cdot 5+\frac{30}{0.4} \cdot \frac{1}{2} \cdot 1 \cdot 6.403\right) \\
& =\alpha \cdot(-7.771+87.49)=79.72 \cdot \alpha \\
& \frac{2.904}{\text { EI }} \cdot \mathrm{X}_{1}+79.72 \cdot \alpha=0 \\
& \mathrm{X}_{1}=-27.4518 \cdot \mathrm{EI} \cdot \alpha=-12.30 \mathrm{kNm}
\end{aligned}
$$

## Displacement Method

## Problem

The structure is the same as the one from problem 1.15


$$
\begin{aligned}
& E I=1.134 \cdot 10^{5} \mathrm{kNm}^{2} \\
& h_{12}=50 \mathrm{~cm} \\
& h_{34}=40 \mathrm{~cm} \\
& h_{23}=h_{35}=60 \mathrm{~cm} \\
& \alpha=10^{-5}
\end{aligned}
$$

The degree of static indeterminacy of the structure is 3 , while the degree of kinematic indeterminacy is 2 . Therefore, for this problem, it is easier to use the displacement method (2 unknowns - one rotation and one linear displacement) instead of the flexibility method (3 unknowns).

Primary structure


Equilibrium equations

$$
\left\{\begin{array}{l}
\mathrm{R}_{1 \mathrm{t}}=\mathrm{r}_{11} \cdot \mathrm{D}_{1}+\mathrm{r}_{12} \cdot \mathrm{D}_{2}+\mathrm{R}_{1 \mathrm{t}}^{0}=0 \\
\mathrm{R}_{2 \mathrm{t}}=\mathrm{r}_{21} \cdot \mathrm{D}_{1}+\mathrm{r}_{22} \cdot \mathrm{D}_{2}+\mathrm{R}_{2 \mathrm{t}}^{0}=0
\end{array}\right.
$$

$r_{11}$ - moment reaction in the rotational restraint 1 , caused by the unit rotation of $D_{1}$ imposed on the primary structure;
$r_{22}$ - force reaction in the linear displacement restraint 2, caused by the unit displacement of $D_{2}$ imposed on the primary structure;
$r_{12}$ - moment reaction in the rotational restraint 1 , caused by the unit displacement of $D_{2}$ imposed on the primary structure;
$r_{21}$ - force reaction in the linear displacement restraint 2, caused by the unit rotation of $D_{1}$ imposed on the primary structure;
$\mathrm{R}_{1 t}^{0}$ - moment reaction in the rotational restraint 1, caused by the temperature variation on the primary structure;
$\mathrm{R}_{2 t}^{0}$ - force reaction in the linear displacement restraint 2 caused by the temperature variation on the primary structure;
$R_{1 \mathrm{t}}$ - moment reaction in the rotational restraint 1 , caused by the rotation $D_{1}$, linear displacement $D_{2}$ and the temperature variation on the primary structure;
$R_{2 t}$ - force reaction in the linear displacement restraint 2, caused by the rotation $D_{1}$, linear displacement $D_{2}$ and the temperature variation on the primary structure.

Fixed end moments on the primary structure due to $t_{m}$


$$
\left\{\begin{array}{l}
\Delta \mathrm{l}_{12}=\alpha \cdot \mathrm{t}_{\mathrm{m} 12} \cdot \mathrm{l}_{12}=\alpha \cdot 25 \cdot 4=100 \cdot \alpha \\
\Delta \mathrm{l}_{23}=\alpha \cdot \mathrm{t}_{\mathrm{m} 24} \cdot \mathrm{l}_{24}=\alpha \cdot 25 \cdot 5=125 \cdot \alpha \\
\Delta \mathrm{l}_{34}=\alpha \cdot \mathrm{t}_{\mathrm{m} 34} \cdot \mathrm{l}_{34}=\alpha \cdot 15 \cdot 4=60 \cdot \alpha \\
\Delta \mathrm{l}_{35}=\alpha \cdot \mathrm{t}_{\mathrm{m} 45} \cdot \mathrm{l}_{45}=\alpha \cdot 25 \cdot 4=100 \cdot \alpha
\end{array}\right.
$$

$$
\mathrm{M}_{12}^{\mathrm{tm}}=3 \cdot \frac{\mathrm{E} \cdot 2 \mathrm{I}}{\left(\mathrm{l}_{12}\right)^{2}} \cdot\left(\Delta \mathrm{l}_{23}+\Delta \mathrm{l}_{35}\right)=95.68 \mathrm{kNm}
$$

$$
\mathrm{M}_{32}^{\mathrm{tm}}=3 \cdot \frac{\mathrm{E} \cdot 3 \mathrm{I}}{\left(\mathrm{l}_{23}\right)^{2}} \cdot\left(\Delta \mathrm{l}_{12}-\Delta \mathrm{l}_{34}\right)=16.33 \mathrm{kNm}
$$

$$
\mathrm{M}_{43}^{\mathrm{tm}}=\mathrm{M}_{34}^{\mathrm{tm}}=\frac{6 \cdot \mathrm{EI}}{\left(\mathrm{l}_{34}\right)^{2}} \cdot \Delta \mathrm{l}_{35}=42.53 \mathrm{kNm}
$$

$$
\mathrm{M}_{35}^{\mathrm{tm}}=3 \cdot \frac{\mathrm{E} \cdot 3 \mathrm{I}}{\left(\mathrm{l}_{35}\right)^{2}} \cdot \Delta \mathrm{l}_{34}=38.27 \mathrm{kNm}
$$



Fixed end moments on the primary structure due to $\Delta t$

$$
\mathrm{M}_{12}^{\Delta \mathrm{t}}=\frac{3}{2} \cdot \alpha \cdot \frac{\Delta \mathrm{t}}{\mathrm{~h}_{12}} \cdot \mathrm{E} \cdot 2 \mathrm{I}=\frac{3}{2} \cdot 10^{-5} \cdot \frac{20}{0.5} \cdot 2 \cdot 1.134 \cdot 10^{5}=136.08
$$

$$
\begin{gathered}
\mathrm{M}_{32}^{\Delta \mathrm{t}}=\frac{3}{2} \cdot \alpha \cdot \frac{\Delta \mathrm{t}}{\mathrm{~h}_{24}} \cdot \mathrm{E} \cdot 3 \mathrm{I}=\frac{3}{2} \cdot 10^{-5} \cdot \frac{20}{0.6} \cdot 3 \cdot 1.134 \cdot 10^{5}=170.1 \\
\mathrm{M}_{35}^{\Delta \mathrm{t}}=\frac{3}{2} \cdot \alpha \cdot \frac{\Delta \mathrm{t}}{\mathrm{~h}_{45}} \cdot \mathrm{E} \cdot 3 \mathrm{I}=\frac{3}{2} \cdot 10^{-5} \cdot \frac{20}{0.6} \cdot 3 \cdot 1.134 \cdot 10^{5}=170.1 \\
\mathrm{M}_{34}^{\Delta \mathrm{t}}=\mathrm{M}_{43}^{\Delta \mathrm{t}}=\alpha \cdot \frac{\Delta \mathrm{t}}{\mathrm{~h}} \cdot \mathrm{E} \cdot \mathrm{I}=10^{-5} \cdot \frac{10}{0.4} \cdot 1.134 \cdot 10^{5}=28.35 \\
170.1 \\
\text { (2) (4) (4) } \mathrm{M}_{\Delta \mathrm{t}}^{0} \text { (1) }
\end{gathered}
$$

Moment reaction in the restrained joint


$$
\begin{array}{ll}
\mathrm{R}_{1 \mathrm{t}}^{\mathrm{tm}}+38.27-42.53+16.33=0 & \Rightarrow \mathrm{R}_{1 \mathrm{t}}^{\mathrm{tm}}=-12.07 \\
\mathrm{R}_{1 \mathrm{t}}^{\Delta \mathrm{t}}+170.1+28.35-170.1=0 & \Rightarrow \mathrm{R}_{1 \mathrm{t}}^{\Delta \mathrm{t}}=-28.35 \\
& \\
\mathrm{R}_{1 \mathrm{t}}^{0}=\mathrm{R}_{1 \mathrm{t}}^{\mathrm{tm}}+\mathrm{R}_{1 \mathrm{t}}^{\Delta \mathrm{m}}=-40.42 \mathrm{kNm}
\end{array}
$$

Force reaction in the restrained joint

$$
\begin{aligned}
& =1 \\
& \left(\sum_{9} \mathrm{M}\right)_{1}=0 \quad-\mathrm{R}_{\mathrm{V}_{21}}^{\mathrm{tm}} \cdot 4+95.68=0 \quad \mathrm{R}_{\mathrm{V} 21}^{\mathrm{tm}}=23.92 \mathrm{kN} \\
& \left(\sum_{\mathrm{R}}^{\mathrm{M}}\right)_{3}^{\mathrm{R}}=0 \quad-\mathrm{R}_{\mathrm{V} 43}^{\mathrm{tm}} \cdot 4+42.53 \cdot 2=0 \quad \mathrm{R}_{\mathrm{V} 43}^{\mathrm{tm}}=21.27 \mathrm{kN} \\
& \mathrm{R}_{2}^{\mathrm{tm}}+23.92+21.27=0 \quad \mathrm{R}_{2}^{\mathrm{tm}}=-45.19 \mathrm{kN}
\end{aligned}
$$

$$
\begin{aligned}
& R_{V 21}^{\Delta t} \cdot 4-136.02=0 \\
& \left(\sum_{2} M\right)_{1}=0 \quad R_{V 21}^{\Delta t}=34.02 \\
& \left(R_{V 21}^{\Delta t}=34.02\right.
\end{aligned} R_{3}^{\Delta t}-34.02 \mathrm{kN}
$$

The fixed end moments due to unit rotation $D_{1}=1$ and unit linear displacement $D_{2}=1$ are the same as for the structure from problem 1.16

$$
\begin{aligned}
& \mathrm{r}_{11}=5.05 \cdot \mathrm{EI} \\
& \mathrm{r}_{12}=-0.375 \cdot \mathrm{EI}=\mathrm{r}_{21} \\
& \mathrm{r}_{22}=0.2814 \cdot \mathrm{EI} \\
& \left\{\begin{array}{l}
5.05 \cdot \mathrm{D}_{1}+(-0.375) \cdot \mathrm{D}_{2}+(-40.42)=0 \\
-0.375 \cdot \mathrm{D}_{1}+0.2814 \cdot \mathrm{D}_{2}+(-11.17)=0
\end{array}\right. \\
& \mathrm{D}_{1}=\frac{12.154}{\mathrm{EI}} \\
& \mathrm{D}_{2}=\frac{55.892}{\mathrm{EI}}
\end{aligned}
$$

# 4. STATICALLY INDETERMINATE STRUCTURES SUBJECTED TO SUPPORT SETTLEMENTS 

## Flexibility Method

## Problem

The structure is the same as the one from problem 1.3 and 2.1

$E I=4.48 \cdot 10^{4} \mathrm{kNm}^{2}$ $\tan \alpha=\frac{5}{4}$ $\alpha=51.34^{\circ}$ $\sin \alpha=0.781$ $\cos \alpha=0.625$

$$
v_{1}=1 \mathrm{~cm} \prod_{\Delta}^{\Delta^{1}}
$$



Primary structure


Compatibility equation

$$
\mathrm{D}_{1 \mathrm{~s}}=\mathrm{d}_{11} \cdot \mathrm{X}_{1}+\mathrm{D}_{1 \mathrm{~s}}^{0}=+\varphi_{5}
$$

$d_{11}$ - rotation at the point and in the direction of the redundant moment reaction $X_{1}$, caused by the unit value of $X_{1}$, acting on the primary structure;
$D_{1 s}^{0}$ - rotation at the point and in the direction of the redundant moment reaction $X_{1}$, caused by the displacements of the supports on the primary structure;
$D_{1 s}$ - rotation at the point and in the direction of the redundant moment reaction $X_{1}$, caused by $X_{1}$ and by the displacements of the supports on the primary structure; this must be identical to the rotation of the support on the real structure $\left(\varphi_{5}\right)$.
(2) $\underbrace{(3)}_{T_{\mathrm{R}_{\mathrm{V} 1}}^{1}=0.111}$

$\mathrm{d}_{11}=\frac{2.904}{E I}$
$\mathrm{D}_{1 \mathrm{~s}}^{0}=-\left[\mathrm{R}_{\mathrm{V} 1}^{1} \cdot\left(-\mathrm{v}_{1}\right)+\left(-\mathrm{R}_{\mathrm{V} 5}^{1}\right) \cdot\left(-\mathrm{V}_{5}\right)+\left(-\mathrm{R}_{\mathrm{H} 5}^{1}\right) \cdot \mathrm{u}_{5}\right]$

$$
=0.111 \cdot(-0.01)+0.111 \cdot(0.02)+(-0.111) \cdot(0.015)
$$

$$
=0.000555
$$

$$
\varphi_{5}=1.2^{\circ}=1.2 \cdot \frac{\pi}{180} \mathrm{rad}=0.020944
$$

$$
D_{1 s}=\frac{2.904}{E I} X_{1}+0.000555=0.020944
$$

$$
\mathrm{X}_{1}=70.21 \cdot 10^{-4} \mathrm{EI}=314.54 \mathrm{kNm}
$$

### 3.2 Displacement Method

## Problem

The structure is the same as the one from problem 1.15 and 2.2.


$$
E I=1.134 \cdot 10^{5} \mathrm{kNm}^{2}
$$

$$
\mathrm{v}_{1}=0.6 \mathrm{~cm} \square_{-1}^{\text {and } 11}
$$

$$
\varphi_{4}=0.8^{\circ}
$$

$$
\mathrm{v}_{5}=1 \mathrm{~cm} \square_{\square}^{\Delta^{5}}
$$

Primary structure


Equilibrium equations

$$
\left\{\begin{array}{l}
R_{1 s}=r_{11} \cdot D_{1}+r_{12} \cdot D_{2}+R_{1 s}^{0}=0 \\
R_{2 s}=r_{21} \cdot D_{1}+r_{22} \cdot D_{2}+R_{2 s}^{0}=0
\end{array}\right.
$$

$r_{11}$ - moment reaction in the rotational restraint 1 , caused by the unit rotation of $D_{1}$ imposed on the primary structure;
$r_{22}$ - force reaction in the linear displacement restraint 2, caused by the unit displacement of $D_{2}$ imposed on the primary structure;
$r_{12}$ - moment reaction in the rotational restraint 1 , caused by the unit displacement of $D_{2}$ imposed on the primary structure;
$r_{21}$ - force reaction in the linear displacement restraint 2, caused by the unit rotation of $D_{1}$ imposed on the primary structure;
$R_{1 s}^{0}$ - moment reaction in the rotational restraint 1 , caused by the displacement of the support on the primary structure;
$R_{2 s}^{0}$ - force reaction in the linear displacement restraint 2, caused by the displacement of the support on the primary structure;
$R_{1 \mathrm{~s}}$ - moment reaction in the rotational restraint 1, caused by the rotation $D_{1}$, linear displacement $D_{2}$ and the displacement of the support on the primary structure;
$R_{2 s}$ - force reaction in the linear displacement restraint 2, caused by the rotation $D_{1}$, linear displacement $D_{2}$ and the displacement of the support on the primary structure.

Fixed end moments caused by support displacement


$\varphi_{23}^{\mathrm{v} 4}=\frac{0.004}{5} \quad \mathrm{M}_{32}^{\mathrm{V} 3}=\frac{3 \cdot \mathrm{E} \cdot 3 \mathrm{I}}{5} \cdot \frac{0.004}{5}=0.00144 \cdot \mathrm{EI}$ $\varphi_{35}^{\mathrm{V} 4}=\frac{0.004}{4} \quad \mathrm{M}_{35}^{\mathrm{V} 3}=\frac{3 \cdot \mathrm{E} \cdot 3 \mathrm{I}}{4} \cdot \frac{0.004}{4}=0.00225 \cdot \mathrm{EI}$


$$
\varphi_{34}^{\mathrm{u} 4}=\frac{0.005}{4} \quad \mathrm{M}_{43}^{\mathrm{u} 3}=\mathrm{M}_{34}^{\mathrm{u} 3}=\frac{6 \cdot \mathrm{EI}}{4} \cdot \frac{0.005}{4}=0.001875 \cdot \mathrm{EI}
$$



$$
\begin{aligned}
& \varphi_{4}=0.8^{\circ}=0.8 \cdot \frac{\pi}{180} \\
& \mathrm{M}_{34}^{\varphi_{4}}=\frac{4 \cdot \mathrm{EI}}{4} \cdot 0.8 \cdot \frac{\pi}{180}=13.96 \cdot 10^{-3} \cdot \mathrm{EI} \\
& \mathrm{M}_{43}^{\varphi_{4}}=\frac{2 \cdot \mathrm{EI}}{4} \cdot 0.8 \cdot \frac{\pi}{180}=6.98 \cdot 10^{-3} \cdot \mathrm{EI}
\end{aligned}
$$


$\varphi_{35}^{\mathrm{V5}}=\frac{0.01}{4}$
$\mathrm{M}_{45}^{\mathrm{v5}}=\frac{3 \cdot \mathrm{E} \cdot 3 \mathrm{I}}{4} \cdot \frac{0.01}{4}=5.625 \cdot 10^{-3} \cdot \mathrm{EI}$

Final diagram for fixed end moments caused by support displacements


Moment reaction in the restrained joint


$$
\begin{aligned}
& \mathrm{R}_{1 \mathrm{~s}}^{0}+3.375 \cdot 10^{-3} \cdot \mathrm{EI}+5.105 \cdot 10^{-3} \cdot \mathrm{EI}-0.72 \cdot 10^{-3} \cdot \mathrm{EI}=0 \\
& \mathrm{R}_{1 \mathrm{~s}}^{0}=-7.76 \cdot 10^{-3} \cdot \mathrm{EI}=880 \mathrm{kNm}
\end{aligned}
$$

Force reaction in the restrained joint

$$
\begin{aligned}
& \left(\sum \mathrm{M}\right)_{3}=0 \quad \mathrm{R}_{\mathrm{V} 34}^{\mathrm{s}} \cdot 4+(-5.105-12.085) \cdot 10^{-3} \cdot \mathrm{EI}=0 \\
& \mathrm{R}_{\mathrm{V} 34}^{\mathrm{s}}=\frac{17.19}{4} \cdot 10^{-3} \cdot \mathrm{EI}=4.2975 \cdot 10^{-3} \cdot \mathrm{EI} \\
& \mathrm{R}_{2 \mathrm{~s}}^{0}-\mathrm{R}_{\mathrm{V} 34}^{\mathrm{s}}=0 \quad \mathrm{R}_{2 \mathrm{~s}}=\mathrm{R}_{\mathrm{V} 34}^{\mathrm{s}}=4.2975 \cdot 10^{-3} \cdot \mathrm{EI}
\end{aligned}
$$

The fixed end moments due to unit displacement $D_{1}$ and unit linear displacement $D_{2}$ are the same as for the structure from problem 1.15.

$$
\begin{aligned}
& \mathrm{r}_{11}=5.05 \mathrm{EI} \\
& \mathrm{r}_{12}=-0.375 \mathrm{EI} \\
& \mathrm{r}_{22}=0.2814 \mathrm{EI}
\end{aligned}
$$

$$
\left\{5.05 \mathrm{EI} \cdot \mathrm{D}_{1}-0.375 \mathrm{EI} \cdot \mathrm{D}_{2}-7.76 \cdot 10^{-3} \cdot \mathrm{EI}=0\right.
$$

$$
\left\{-0.375 \mathrm{EI} \cdot \mathrm{D}_{1}+0.2813 \mathrm{EI} \cdot \mathrm{D}_{2}+4.275 \cdot 10^{-3} \cdot \mathrm{EI}=0\right.
$$

$$
\mathrm{D}_{1}=0.4462 \cdot 10^{-3}
$$

$$
\mathrm{D}_{2}=-14.683 \cdot 10^{-3}
$$

